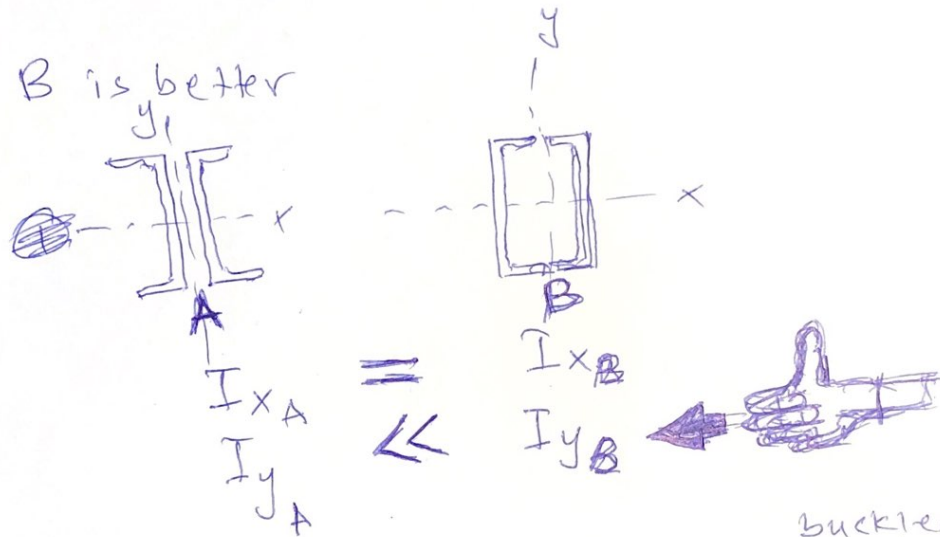


Question 1

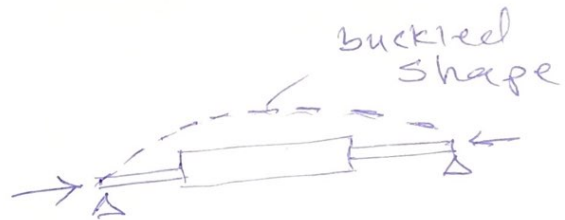
(a) Unless they are short, members fail by buckling (local or overall) ^{compression}



(b) B is better



(c)



- compressed member
- fails by buckling
- Potential buckling produces largest displacement, and hence, largest moment in the middle segment \Rightarrow Provide larger cross-section to increase I (and A).

(d) ~~rod~~ cable and rod are tension-only members, and are flexible in nature. Applying the requirement $L/r < 300$ (to keep shape) produces unnecessary (unpractical) huge section. These members are typically long diagonals.

Problem 2

Evaluate full tensile capacity of W-shape

1. Yield criterion:

$$\phi_t T_n = 0.9(50)(11.2) = \boxed{504} \text{ KIPS}$$

2. Rupture

$$\phi_t T_n = 0.75 F_u A_e$$

$$A_n = 11.2 - 4\left(\frac{7}{8} + \frac{1}{8}\right) \times 0.515 = 9.14 \text{ W14 X 38, } A = 11.2$$

$$\frac{\text{width}}{\text{depth}} < \frac{2}{3} d \Rightarrow U = 0.85$$

$$\phi_t T_n = 0.75(65)(0.85 \times 9.14) = \boxed{378.74} \text{ KIPS}$$

3. Block shear

Take spacing and edge distances = $2 \frac{2}{3} d_b = 2.333 \text{ in.}$

$$R_n = 0.6 F_u A_{nv} + U_b F_u A_{nt} \leq 0.6 F_y A_{gv} + U_b F_u A_{nt}$$

$$A_{gv} = [4(2.333) \times (0.515)] \times 4 = 19.612 \text{ in}^2$$

$$A_{nv} = 19.612 - [3.5\left(\frac{7}{8} + \frac{1}{8}\right) \times (0.515)] \times 4 = 12.4 \text{ in}^2$$

$$A_{nt} = [2.333 - 0.5\left(\frac{7}{8} + \frac{1}{8}\right)] \times (0.515) \times 4 = 3.78 \text{ in}^2$$

Subst.

$$R_n = 0.6(65)(12.4) + 1.0(50)(3.78) \leq 0.6(50)(19.612) + 1.0(65)(3.78)$$

$$672.6 < 834.06 \text{ KIPS}$$

$$\phi R_n = 0.75(672.6) = \boxed{504.45} \text{ KIPS}$$

\Rightarrow Rupture criterion controls $\phi_t T_n = 378.74$

$$\text{For one plate } (\phi_t T_n)_{\text{plate}} = \frac{378.74}{2} = 189.37 \text{ KIP}$$

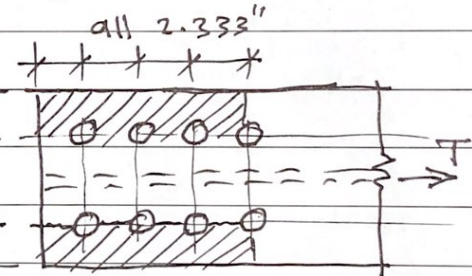
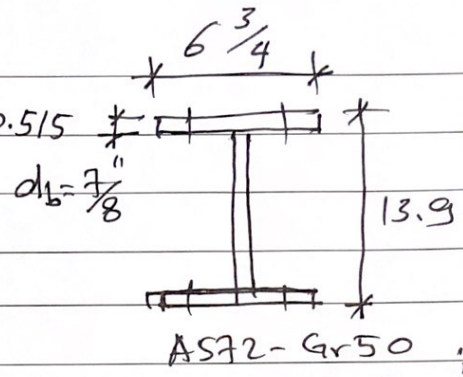
For the plate:

1. Yield: $\phi_t T_n \geq 0.9(50)(8 \times t) = 189.37 \Rightarrow t \geq 0.526 \text{ in.}$

2. Rupture $\phi_t T_n \geq 0.75(65)(1.0) \times [8 - 2\left(\frac{7}{8} + \frac{1}{8}\right)] \times t = 189.37 \Rightarrow t \geq 0.65 \text{ in.}$

$$\Rightarrow \text{Use } t_p = 0.65 \text{ in}$$

3. Block shear is OK, since similar mode as W shape but with larger width and thickness.



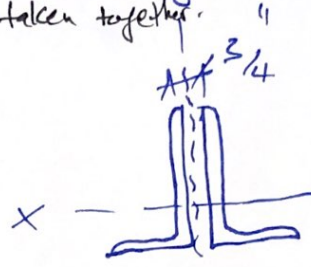
Problem 3

⇒ Not like ~~compression~~ tension member
The two angles must be taken together.

$$A = 2.4 \text{ in}^2$$

$$r_x = 0.953$$

$$r_y = 1.03$$



2L3x2x1/4 L2BB

A36

$$A = 2.4 \text{ in}^2$$

$$I_x =$$

$$I_y =$$

Truss member ⇒ $k_x = k_y = 1$

$$\frac{k_x L_x}{r_x} = \frac{(1)(6 \times 12)}{0.953}$$

$$\frac{k_y L_y}{r_y} = \frac{(1.0)(6 \times 12)}{1.03}$$

$$75.55 > 69.9$$

X-axis controls

$$4.71 \sqrt{\frac{E}{F_y}} = 4.71 \sqrt{\frac{29000}{36}} = 133.68 < \frac{L_x}{r_x}$$

⇒ Member will fail by inelastic buckling

$$\Rightarrow F_{cr} = 0.658 \left(\frac{F_y}{F_e}\right) * F_y$$

$$F_e = \frac{\pi^2 E}{\left(\frac{k_x L_x}{r_x}\right)^2} = \frac{\pi^2 (29000)}{(75.55)^2} = 50.14$$

$$F_{cr} = \left(0.658\right)^{\frac{36}{50.14}} * 36 = 26.63 \text{ ksi}$$

$$\phi_c P_n = 0.9 (26.63) (2.4) = 57.52 \text{ k}$$

~~⇒ P = 57.52 k~~

$$P_u = 57.52 \text{ kips}$$